

STUDENTS: Dane Bowman-Weston, Kyle Jiang, Kamiar Pousti, Sakar Shakya, Cameron Urquhart, John Viacrucis

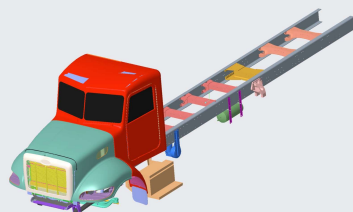
## SUMMARY

### Introduction:

- The goal of this 4-year project is to convert a Class 7 Peterbilt 337 ICE truck into a fully battery electric vehicle.
- We are working closely with 3 other E-Truck capstone teams: **Controls** Architecture, **Electrical** Architecture, and **Systems** Definition & Modeling.
- We also collaborate with the E-Truck Registered Student Organization (RSO).

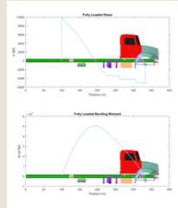
### Objective:

- Select appropriate components based on industry performance metrics using Market Research, Six Sigma Decision tools, and Supplier consultations.
- Ensure structural integrity and safe mounting of electrical components through Static Beam Analysis and Finite Element Analysis (FEA), including Modal Analysis.
- Modify CAD model to package battery-electric powerplant into the ladder chassis.
- Modify auxiliary systems to operate without ICE power



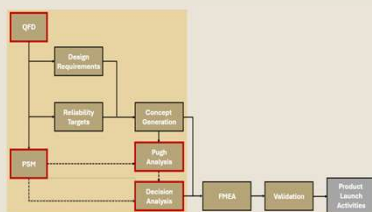
## STRUCTURAL ANALYSIS

Static beam equations were used to find the shear and bending moment of one frame rail in the chassis at a time, as well as to find how the front and rear axles were loaded. Using the concept of superposition, each load case, such as the weight of the frame and of the cabin, was analyzed separately and added together into one graph. After the battery loads were determined, the remaining weight that would bring the total to 33,000 lbs. (Class 7 limit) was derived and placed as its own load located on the back of the truck. The resulting graph is the distribution of 33,000 lbs. on the truck chassis. The maximum shear on one frame rail is ~9,866 lbs. while the maximum bending moment is ~501,400 in-lb.



## DECISION ANALYSIS

This Process Flowchart is what PACCAR uses

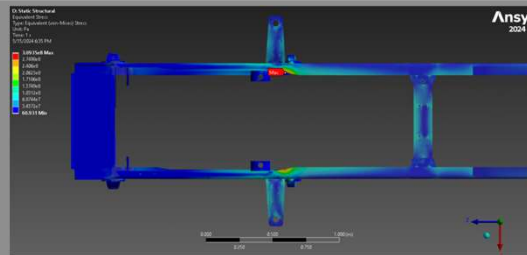


**E-Axle TOPSIS**

Criteria	Weight	Value	Normalized Value	Weighted Value
Efficiency	0.25	90	0.15	0.0375
Reliability	0.20	85	0.14	0.028
Cost	0.15	75	0.10	0.015
Maintainability	0.10	80	0.08	0.008
Performance	0.10	95	0.13	0.013
Flexibility	0.10	85	0.10	0.010
Quality	0.10	90	0.10	0.010
Supportability	0.10	80	0.08	0.008

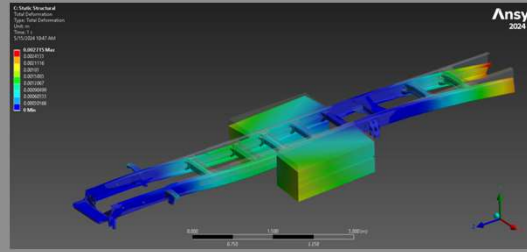
## 6G CABIN STRESS

Stress incurred at the site of the truck cabin due to a scenario in which the driver reverses into a loading dock at 6G's (dock strike). Maximum stress in this area is ~ 309 MPa



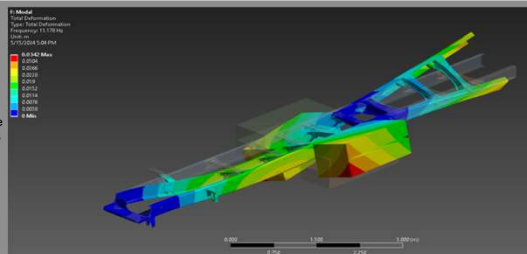
## 6G DEFORMATION

Image is an exaggeration of the deformation in the frame due to 6G dock strike. Max deformation is ~ 3.8 mm



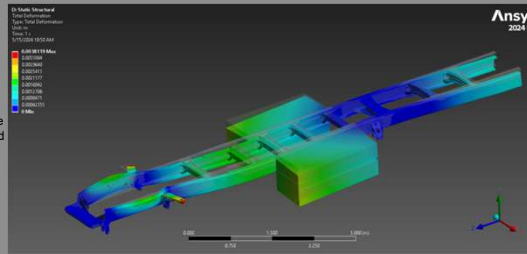
## MODAL ANALYSIS

In Modal ANSYS, the first 6 natural frequencies (modes) of the frame were found and ranged from 11.18 to 25.33 Hz. Harmonic Response analysis suggests the frame reaches extreme amplitudes at 11.18 Hz



## 2G FATIGUE

Image is an exaggeration of the deformation in the frame due to fatigue analysis of 2G bump and 1.5G rebound in the suspension. Current model suggests infinite life in the ladder chassis with batteries mounted



## COMPONENT SELECTION

	DANA—Spicer Zero-8 E-Axle & Inverter	
	Power: 530 kW	Voltage: 400-800 V
	AKASOL—9 AKM 150 CYC HV Battery (x4)	
	Torque: 28,000-130,000 Nm	GCW: 16,000-70,000 kg
	AirSquared—PT7H043D-BLDC-LC Air Compressor	
	Energy: 98 kWh	Weight: 560 kg
	Bell Intercoolers—EWP130 Water Pump	
	Voltage: 665 V	Cycles: 4,000
	DC Airco—DC 6001 Heat Pump	
	Pressure: 190 psi	Flow: 425 L/min
	GUCHEN—Electric Truck AC Compressor	
	Displacement: 134.5 cm³/rev	Weight: 18 kg
	Bell Intercoolers—EWP130 Water Pump	
	Pressure: 72.5 psi	Flow: 130 L/min
	DC Airco—DC 6001 Heat Pump	
	Speed: 600-1200 rpm	Weight: 1.226 kg
	GUCHEN—Electric Truck AC Compressor	
	Power: 3.4 kW	Voltage: 400-850 V
	GUCHEN—Electric Truck AC Compressor	
	Flow: 25-40 L/min	Weight: 21.5 kg
	GUCHEN—Electric Truck AC Compressor	
	Power: 2.82 kW	Cooling: 3.175 kW
	GUCHEN—Electric Truck AC Compressor	
	Displacement: 34 cm³/rev	Weight: 6.1 kg

## CONCLUSION

### Summary:

Throughout the two quarters, our team made substantial progress. We conducted comprehensive market research based on the requirements developed in collaboration with the 3 other capstone teams. Through supplier consultations and the application of Six Sigma tools like the Decision Analysis and the Pugh Matrix, we generated rankings for all components. We successfully analyzed the chassis structure to ensure structural integrity and safe mounting of the high voltage batteries. Our team also modified the CAD model to incorporate the selected high-voltage batteries.

### Future Work:

- Finalize the selection of all components and maintain ongoing supplier engagement
- Conduct a Decision Analysis to affirm the High Voltage Battery selection
- Confirm decisions regarding the necessarily auxiliary components for the cooling system
- Refine FEA, including Modal analysis with all components selected
- Update the CAD model to incorporate the E-axle, auxiliary components, and component mounting brackets

## ACKNOWLEDGEMENTS

- Faculty Advisor(s): Per Reinhall
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